Author(s): Tuovinen, Toni & Hinkkanen, Marko & Luomi, Jorma
Title: Analysis and design of a position observer with resistance adaptation for synchronous reluctance motor drives
Year: 2011
Version: Post print

Please cite the original version:

Rights: © 2011 Institute of Electrical & Electronics Engineers (IEEE). Permission from IEEE must be obtained for all other uses, in any current or future media, including reprinting/republishing this material for advertising or promotional purposes, creating new collective works, for resale or redistribution to servers or lists, or reuse of any copyrighted component of this work in other work.
Analysis and Design of a Position Observer with Resistance Adaptation for Synchronous Reluctance Motor Drives

Toni Tuovinen, Marko Hinkkanen, and Jorma Luomi

Aalto University School of Electrical Engineering
Department of Electrical Engineering, P.O. Box 13000, FI-00076 Aalto, Finland

Abstract—A back-EMF-based reduced-order position observer with stator-resistance adaptation is analyzed for motion-sensorless synchronous reluctance motor drives. Analytical equations for steady-state estimation errors and stability conditions are derived (with and without resistance adaptation), taking into account errors in the parameter estimates. The effect of the observer gain on the noise reduction is studied by means of eigenvector analysis. A robust gain selection is proposed, which maximizes the allowed uncertainties in the parameter estimates. The proposed observer design is experimentally evaluated using a 6.7-kW synchronous reluctance motor drive; stable operation is demonstrated at low speeds under various parameter errors.

Index Terms—Observer, stability conditions, speed sensorless, stator resistance estimation.

I. INTRODUCTION

Modern synchronous reluctance motors (SYRs) are becoming interesting competitors to induction motors and permanent-magnet synchronous motors in variable-speed drives [1], [2]. The rotor position of a synchronous motor has to be known with good accuracy in order to obtain stable operation and high performance. The rotor position can be either measured or estimated. Motion-sensorless control is usually preferable: motion sensors are expensive, they can be damaged or, in some environments and applications, cannot be installed.

Signal-injection-based methods can be used for SYRMs. In order to avoid additional noise and losses, it is desirable to use a method based on the back electromotive force (EMF), and combine a signal-injection method with it only at the lowest speeds [3], [4]. Back-EMF-based observers for SYRM drives have been proposed in [1], [5]–[8]. However, the stability analysis has been omitted, with the exception of [8]. The effect of parameter uncertainties should be accounted for in the stability analysis, which makes the task even more complicated. The model parameters are rarely known accurately, and in practice, they are not constant. The stator resistance varies with the winding temperature during the operation of the motor. The d-axis flux component usually saturates strongly as a function of the d-axis current component, and the d-axis saturation is coupled with the q-axis saturation [9].

In this paper, a reduced-order observer with stator-resistance adaptation—originally proposed for PMSM drives in [10]—is applied for SYRM drives. Analytical equations for steady-state estimation errors and stability conditions are derived, taking into account errors in the parameter estimates (that were omitted in the analysis in [10]). Based on these design tools, a robust gain selection is proposed, which maximizes the allowed uncertainties in the parameter estimates. Furthermore, the effect of the observer gain on the noise reduction is studied by means of eigenvector analysis. If desired, the observer could be augmented with a signal-injection method, for example in a fashion similar to [3]. The performance of the proposed observer design is evaluated using laboratory experiments with a 6.7-kW SYRM drive.

II. SYRM MODEL AND ROTOR-POSITION OBSERVER

A. Model

Real space vectors will be used here. For example, the stator-current vector is \( \hat{i}_s = [\hat{i}_d, \hat{i}_q]^T \), where \( \hat{i}_d \) and \( \hat{i}_q \) are the components of the vector and the matrix transpose is marked with the superscript T. The identity matrix and the orthogonal rotation matrix are defined as

\[
I = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}, \quad J = \begin{bmatrix} 0 & -1 \\ 1 & 0 \end{bmatrix}.
\]

The electrical position of the d axis is denoted by \( \vartheta_m \). The d axis is defined as the direction of the maximum inductance of the rotor. The position depends on the electrical angular rotor speed \( \dot{\vartheta}_m \) according to

\[
\frac{d\vartheta_m}{dt} = \omega_m \tag{1a}
\]

To simplify the analysis in the following sections, the machine model will be expressed in the estimated rotor reference frame, whose d axis is aligned at \( \hat{\vartheta}_m \) with respect to the stator reference frame. The stator inductance is

\[
L = e^{-\hat{\vartheta}_m J} \begin{bmatrix} L_d & 0 \\ 0 & L_q \end{bmatrix} e^{\hat{\vartheta}_m J} \tag{1b}
\]

where \( \hat{\vartheta}_m = \vartheta_m - \hat{\vartheta}_m \) is the estimation error in the rotor position, \( L_d \) the direct-axis inductance, and \( L_q \) the quadrature-axis inductance. The voltage equation is

\[
\frac{d\psi_s}{dt} = u_s - R_s \hat{i}_s - \omega_m J \hat{\vartheta}_m \tag{1c}
\]
where \( \psi_s \) is the stator-flux vector, \( u_s \) the stator-voltage vector, \( R_s \) the stator resistance, and \( \dot{\vartheta}_m \) is \( d\vartheta_m/dt \) is the angular speed of the coordinate system. The stator current is a nonlinear function
\[
i_s = L^{-1}\psi_s
\]
(1d)
of the stator-flux vector and the position error \( \dot{\vartheta}_m \).

B. Observer

A typical rotor-oriented control system is depicted in Fig. 1, where the reduced-order observer proposed in [10] is considered. It is based on estimating the rotor position and the \( d \) component \( \dot{\psi}_d \) of the stator flux in estimated rotor coordinates. For a SyRM, the componentwise presentation of the observer is
\[
\frac{d\dot{\psi}_d}{dt} = u_d - \dot{R_s}i_d + \dot{\omega}_m L_q i_q + k_1(\dot{\psi}_d - \dot{L}_d i_d)
\]
(2a)
\[
\frac{d\dot{\vartheta}_m}{dt} = u_q - \dot{R_s}i_q - \dot{L}_q \frac{di_d}{dt} + k_2(\dot{\psi}_d - \dot{L}_d i_d)
\]
(2b)
where \( \dot{R_s} \), \( \dot{L}_d \) and \( \dot{L}_q \) are estimates of the corresponding actual parameters, and \( k_1 \) and \( k_2 \) are observer gains. The observer is of the second order and there are only two gains.

With accurate parameter estimates, the closed-loop system consisting of (1) and (2) is locally stable in every operating point if the gains are given by
\[
k_1 = -\frac{b + \beta(c/\dot{\omega}_m - \dot{\omega}_m)}{\beta^2 + 1}, \quad k_2 = \frac{\beta b - c/\dot{\omega}_m + \dot{\omega}_m}{\beta^2 + 1}
\]
(3)
where \( \beta = i_q/i_d \) and the design parameters \( b > 0 \) and \( c > 0 \) may depend on the operating point\(^1\). The observer gain design problem is reduced to the selection of the two positive parameters \( b \) and \( c \), which are actually the coefficients of the characteristic polynomial of the linearized closed-loop system. Hence, (3) can be used to place the poles of the linearized closed-loop system arbitrarily.

\(^1\)For \( \dot{\omega}_m = 0, c = 0 \) has to be selected to avoid division by zero, giving only marginal stability for zero speed.

For improved robustness at the lowest speeds, the observer (2) can be augmented with the stator-resistance adaptation law [10]
\[
\frac{d\dot{R}_s}{dt} = k_R(\dot{\psi}_d - \dot{L}_d i_d)
\]
(4)
where \( k_R \) is the adaptation gain. With accurate parameter estimates, the general stability conditions for the system augmented with (4) are
\[
k_R(i_d - \beta i_q) b - 2 i_q \dot{\omega}_m + bc > 0
\]
(5b)
where \( b \) and \( c \) are the positive design parameters in (3).

III. SELECTION OF GAINS

With parameter errors included, the stability is not guaranteed for all positive values of the design parameters \( b \) and \( c \) in (3). It will be studied how these design parameters should be chosen in order to reduce sensitivity to parameter errors (Section III-A) and process noise (Section III-B).

The nonlinear estimation error dynamics of the closed-loop system consisting of (1) and (2) are
\[
\frac{d\dot{\psi}_s}{dt} = (K \dot{L} - \dot{\omega}_m J)\dot{\psi}_s + [K(\dot{L}^{-1} L - I) - \tilde{R}_s] i_s
\]
(6a)
where the gain matrix is
\[
K = \begin{bmatrix} k_1 \dot{L}_d & 0 \\ \frac{b}{L_d} \dot{L}_d & 0 \end{bmatrix}
\]
(6b)
The estimation error of the stator flux is \( \dot{\psi}_s = \dot{\psi}_s - \dot{\psi}_s \) and other estimation errors are defined similarly. In the reduced-order observer (2), the error of the \( q \)-axis flux is a nonlinear function of the position error, since \( \psi_q = \dot{L}_q i_q \) and
\[
\psi_q = \frac{1}{2}(L_q - L_d) \left( \sin(2\dot{\vartheta}_m) i_d + \cos(2\dot{\vartheta}_m) i_q \right)
\]
(6c)
\[
+ \frac{1}{2}(L_d + L_q) i_q.
\]
A. Stability With Uncertain Parameters

With erroneous parameter estimates, the stability of (6) should be analyzed in the vicinity of the steady-state position error \( \dot{\vartheta}_{m0} \) that is nonzero. It is assumed that the estimation error consists of the steady-state error and a small variation in the vicinity of that steady-state error. In steady-state, the time derivative of \( \dot{\psi}_s \) in (6) vanishes. The equation for the steady-state position error \( \dot{\vartheta}_{m0} \) is
\[
A \cos 2\dot{\vartheta}_{m0} + B \sin 2\dot{\vartheta}_{m0} + C = 0
\]
(7a)
where
\[
A = -\dot{\omega}_m (L_d - L_q) [k_1 + \beta(\dot{\omega}_m - k_2)]
\]
(7b)
\[
B = -\dot{\omega}_m (L_d - L_q) [(\dot{\omega}_m - k_2) - \beta k_1]
\]
(7c)
\[
C = \left[ 2\dot{L}_d - L_d - L_q \right] k_1 \dot{\omega}_m + 2 \dot{R}_s \dot{\omega}_m - k_2 \right] + \left[ 2 \dot{R}_s k_1 + \dot{\omega}_m (L_d + L_q - 2 \dot{L}_q) (\dot{\omega}_m - k_2) \beta
\]
(7d)
The system (6) is locally stable if

\[
\beta' = \tan(2\tilde{\vartheta}_{m0} + \tan^{-1}\beta_0) > 0
\]  

where \(\beta' = k_{20}\beta' - k_{10} > 0\) \(\omega_{m0} = \omega_{m0}(L_d - L_q)\) \(L_d = 0.397 \text{ p.u.}, L_q = 0.05 \text{ p.u.}, i_q = 0.4 \text{ p.u.}, i_{q_0} = 0\). The worst-case stability boundaries corresponding to the parameter uncertainties of 10% and 20% are shown by solid lines.

Fig. 2. Stability map in the design parameter space. All \(b > 0\) and \(c > 0\) guarantee stable operation with the accurate parameter estimates. The operating point corresponds to \(\omega_{m} = 0.05 \text{ p.u.}, i_d = 0.4 \text{ p.u.}, i_q = 0\). The worst-case stability boundaries corresponding to the parameter uncertainties of 10% and 20% are shown by solid lines.

The solution for the steady-state position error is

\[
\tilde{\vartheta}_{m0} = -\sin^{-1}\left(\frac{\hat{G}}{2}\right) + \phi
\]  

where

\[
\phi = \tan^{-1}\left(\frac{A}{B}\right), \quad D = \frac{B}{\cos\phi}
\]

If the steady-state error of the \(d\) axis flux is of interest, it can be obtained from

\[
\dot{\psi}_{d0} = \frac{2k_{20}(2\tilde{\vartheta}_{m0})(L_d - L_q)}{2(k_{20} - \omega_{m0})}
\]

\[
\dot{i}_d R_s + \frac{2(k_{20} - \omega_{m0})}{2(k_{20} - \omega_{m0})} \left[(L_d - L_q) \cos(2\tilde{\vartheta}_{m0}) + L_d + L_q - 2\hat{L}_d\right]
\]

after \(\tilde{\vartheta}_{m0}\) is solved.

The estimation error dynamics (6) are linearized in the vicinity of this steady-state error \(\tilde{\vartheta}_{m0}\), yielding

\[
\frac{d}{dt}\begin{bmatrix} \dot{\psi}_d \\ \dot{\psi}_q \end{bmatrix} = \begin{bmatrix} k_{10} & -k_{10}\beta' + \omega_{m0} \\ k_{20} - \omega_{m0} & -k_{20}\beta' \end{bmatrix} \begin{bmatrix} \dot{\psi}_d \\ \dot{\psi}_q \end{bmatrix}
\]

where

\[
\beta' = \tan(2\tilde{\vartheta}_{m0} + \tan^{-1}\beta_0)
\]

and operating-point quantities are denoted by the subscript 0.

The system (6) is locally stable if

\[
b' = k_{20}\beta' - k_{10} > 0
\]

\[
c' = \omega_{m0} - \omega_{m0}(k_{20} + k_{10}\beta') > 0
\]

where \(b'\) and \(c'\) are coefficients of the characteristic polynomial of (10a). With accurate parameter estimates, \(\beta' = \tan(\tan^{-1}\beta_0) = \beta\), and (11) leads to (3). Equations (7) and

\[\text{Fig. 3. Areas of stable operation for three different values of } i_q \text{ for the parameter uncertainties of } 10\% \text{ and } \omega_m = 0.05 \text{ p.u.: } i_q = 0 \text{ p.u. (solid line), } i_q = 0.8 \text{ p.u. (dotted line), and } i_q = -0.8 \text{ p.u. (dashed line).}
\]

\[\text{(11) could be used to minimize the steady-state position error, to eliminate the effect of a certain parameter error},^2 \text{ or to maximize the parameter tolerances.}
\]

The conditions (11) are studied in no load operation with actual parameters \(L_d = 2.50 \text{ p.u.}, L_q = 0.397 \text{ p.u.}, \text{ and } R_s = 0.043 \text{ p.u. of a } 6.7\text{kW SyRM. The same relative uncertainty is assumed for all three parameter estimates } \hat{L}_d, \hat{L}_q, \text{ and } \hat{R}_s.
\]

Hence, eight different worst-case combinations, consisting of minimum and maximum values of the parameter estimates, can be formed. For example, if the relative uncertainty is defined to be 20%, one of the worst-case combinations is \(\hat{L}_d = 0.8L_d, \hat{L}_q = 1.2L_q, \text{ and } \hat{R}_s = 0.8R_s.
\]

At each studied operating point, the local stability of the observer was analyzed for all eight worst-case combinations of erroneous parameter estimates. First, the estimation error of the rotor position was analytically solved in steady state. Then, the local stability of this operating point was checked using (11).

The stability of the estimation-error dynamics with erroneous parameter estimates was analyzed for different values of the design parameters \(b\) and \(c\). Fig. 2 shows an example of the stability map in the design-parameter space for the parameter uncertainties of 10% and 20%. In the figure, the vertical axis is scaled with the inverse rotor speed (in order to help the comparison at different speeds). The operating point in Fig. 2 is defined by \(\omega_{m} = 0.05 \text{ p.u.}, i_d = 0.4 \text{ p.u.}, \text{ and } i_q = 0\). It can be seen that the region of \(b\) and \(c\) yielding the stable operation is large even in the case of the parameter uncertainty of 20%. As the parameter uncertainty increases, the stable region shrinks (and disappears if the uncertainty is high enough). The size of the stable region depends on the speed and the current components, but its shape remains approximately unchanged. Based on the analysis results, the

\[\text{^2For example, the choice } k_2 = k_1\beta + \omega_{m0} \text{ eliminates } \hat{R}_s.\]
design parameter $c$ can be chosen as
\begin{equation}
    c = \kappa b |\dot{\omega}_m| + \dot{\omega}_m^2
\end{equation}
where the parameter $\kappa$ should approximately correspond to the slope of the line passing through the centers of the stable regions in the stability maps, cf. Fig. 2 as an example. As the uncertainty increases, eventually with the combination $\dot{L}_d < L_d$ and $\dot{R}_s > R_s$, $b'$ in (11) becomes negative, and with the combination $\ddot{L}_d > L_d$ and $\dot{R}_s < R_s$, $|C/D|$ in (8) is larger than one, suggesting that there is no steady-state solution for that combination. Provided that $b \gg \dot{\omega}_m$, it can be shown that the largest possible parameter uncertainty is achieved with $\kappa = \sqrt{3}$ for SyRMs.

In Fig. 3, the stability map for the parameter uncertainties of 10% is depicted for three different values of $i_q$ in a fashion similar to Fig. 2. The d axis current is $i_d = 0.4$ p.u. The values for $i_q$ are $-0.8$ p.u., 0, and 0.8 p.u., corresponding to the negative rated load, no load, and the rated load. It can be seen that the area of stable operation rotates clockwise as $|i_q|$ increases. It seems that a slightly lower value of $\kappa$ can be used for generator operation than for no-load operation. This result can be used to reduce noise in generator operation.

**B. Noise Reduction**

In this subsection, accurate parameters are considered. The eigenvalues of the linearized estimation-error dynamics are
\begin{equation}
    s_{1,2} = -\frac{b \pm \sqrt{b^2 - 4c}}{2}
\end{equation}
At low speeds, $s_1 \approx -\kappa|\dot{\omega}_m|$ and $s_2 \approx -b + \kappa|\dot{\omega}_m|$. In multiple-input-multiple-output systems, the eigenvalues do not provide all the necessary information considering the system dynamics. The eigenvalues describe the gains in the directions of the corresponding eigenvectors. In order to reduce the noise in the position estimate, the eigenvectors of (10a) are analyzed in the following, and a modification to (12) is proposed.

Provided that $|s_1| \neq |s_2|$, any value of $\dot{\psi}_s$ can be written as $\dot{\psi}_s = \lambda_1 v_1 + \lambda_2 v_2$, where $v_1$ and $v_2$ are the eigenvectors corresponding to $s_1$ and $s_2$, respectively. The equation for the small-signal dynamics (10a) can be written as
\begin{equation}
    \frac{d\dot{\psi}_s}{dt} = s_1 \lambda_1 v_1 + s_2 \lambda_2 v_2
\end{equation}
The normalized eigenvectors are
\begin{align}
    v_1 &= \frac{-1}{\sqrt{(s_2^2 - \dot{\omega}_m^2)(1 + \beta^2)}} \left[ \frac{\beta s_2 - \dot{\omega}_m}{s_2 + \beta \dot{\omega}_m} \right] \\
    v_2 &= \frac{-1}{\sqrt{(s_1^2 - \dot{\omega}_m^2)(1 + \beta^2)}} \left[ \frac{\beta s_1 - \dot{\omega}_m}{s_1 + \beta \dot{\omega}_m} \right]
\end{align}

If $c$ is smaller than $b$, which is the case at low speeds, $|s_1|$ is small as compared to $|s_2|$, corresponding to slow and fast dynamics, respectively. It might be desirable that the d axis component of the estimated stator flux is updated in the faster time scale, and the q axis component (angle) is updated in the slower time scale, because the flux dynamics are faster than the speed dynamics. Hence, $v_1$ should point to the q axis direction and $v_2$ should point to the d axis direction, or close to it.

At some operating points, both eigenvectors may point to the q axis direction. Consequently, a small error in the estimated d axis component is compensated by updating the q axis component. Furthermore, the d component of $\nu_2$ is zero when $\kappa \beta \text{sign}(\dot{\omega}_m) \approx -1$ (since $\beta s_1 \approx -\kappa|\dot{\omega}_m|$). In the regenerating mode, the eigenvector of the faster time scale points to the q axis direction, which means that the rotor angle is updated in the faster time scale.

The fast eigenvector can be rotated more to the d axis direction by decreasing the value of $\kappa$. A lower limit $\kappa_{\text{min}}$ has to be used for $\kappa$ since too small values might lead to instability if the parameters are not accurate enough. Therefore, $\kappa$ in (12) is chosen as
\begin{equation}
    \kappa = \begin{cases} 
    \kappa_{\text{min}}, & \text{if } \sqrt{3} + \beta \text{sign}(\dot{\omega}_m) \leq \kappa_{\text{min}} \\
    \sqrt{3} - |\beta|, & \text{if } \kappa_{\text{min}} < \sqrt{3} + \beta \text{sign}(\dot{\omega}_m) < \sqrt{3} \\
    \sqrt{3}, & \text{otherwise.}
    \end{cases}
\end{equation}
It can be seen that the value of $\kappa$ is decreased in the regenerating mode, and $\kappa$ is constant in the motor mode and no-load.

The tracks of the normalized eigenvectors as $i_q$ changes are depicted in Fig. 4 in two cases. The eigenvectors corresponding to slow and fast dynamics are depicted for $\kappa = \sqrt{3}$ in Figs. 4(a) and 4(b), respectively. The eigenvectors corresponding to slow and fast dynamics are depicted when $\kappa$ is selected according to (16) with $\kappa_{\text{min}} = 0.6$. In Figs. 4(c) and 4(d), respectively. The component $i_q$ changes from $-1$ p.u. to 1 p.u., $b = 5$, $i_d = 0.4$ p.u., and $\dot{\omega}_m = 0.1$ p.u.

**C. Stator-Resistance Adaptation**

The nonlinear estimation error dynamics of the system augmented with stator-resistance adaptation consist of (6) and (4). The steady-state position error $\dot{\nu}_{m0}$ is determined by
\begin{equation}
    A \cos 2\ddot{\nu}_{m0} + B \sin 2\ddot{\nu}_{m0} + C = 0
\end{equation}
where
\begin{align}
    A &= (1 - \beta^2) (L_d - L_q) \\
    B &= -2\beta (L_d - L_q) \\
    C &= (1 + \beta^2) (L_d + L_q) - 2 \left[ \dot{L}_d + \beta^2 \ddot{L}_q \right]
\end{align}
The solution for the steady-state position error is given by (8), unless $|\beta| = 1$ (coefficient $A$ vanishes) or $\beta = 0$ (coefficient $B$ vanishes). If $|\beta| = 1$, the steady-state position error is given by
\begin{equation}
    \text{sign}(2\ddot{\nu}_{m0}) = \text{sign}(\beta) \frac{L_d - \dot{L}_d + L_q - \ddot{L}_q}{L_d - L_q}
\end{equation}
and if $\beta = 0$, the steady-state position error is given by
\begin{equation}
    \cos(2\ddot{\nu}_{m0}) = \frac{2\dot{L}_d - L_d - L_q}{L_d - L_q}
\end{equation}
The steady-state errors $\tilde{\psi}_{d0}$ and $\tilde{R}_{s0}$ can be obtained from

$$\tilde{\psi}_{d0} = -\frac{1}{2} i_d \left[ (L_d - L_q) \cos(2\tilde{\vartheta}_{m0}) + L_d + L_q - 2\dot{L}_d \right] + \frac{1}{2} i_q \sin(2\tilde{\vartheta}_{m0})(L_d - L_q)$$

$$\tilde{R}_{s0} = \frac{L_d + L_q - 2\dot{L}_d + (L_d - L_q) \cos(2\tilde{\vartheta}_{m0})}{2}\tilde{\omega}_m$$

These results can be used to optimize the stator-resistance adaptation gain $k_{R0}$ in a fashion similar to optimization of the design parameters $b$ and $c$. With accurate parameter estimates, the linearized system (22) is locally stable if the stability
conditions (5) are satisfied.

IV. EXPERIMENTAL SETUP AND PARAMETERS

The motion-sensorless control system was implemented in a dSPACE DS1104 PPC/DSP board. A 6.7-kW four-pole SyRM is fed by a frequency converter that is controlled by the DS1104 board. The rated values of the SyRM are: rotational speed 3175 r/min; frequency 105.8 Hz; line-to-line rms voltage 370 V; rms current 15.5 A; and torque 20.1 Nm. The base values for angular speed, voltage, and current are defined as $2\pi \cdot 105.8 \text{ rad/s}$, $\sqrt{2/3} \cdot 370 \text{ V}$, and $\sqrt{2} \cdot 15.5 \text{ A}$, respectively.

A servo motor is used as a loading machine. The rotor speed $\omega_m$ and position $\theta_m$ are measured using an incremental encoder for monitoring purposes. The shaft torque is measured using a Dataflex 22 torque measuring shaft. The total moment of inertia of the experimental setup is $0.015 \text{ kgm}^2$ (2.7 times the inertia of the SyRM rotor).

The block diagram of the motion-sensorless control system implemented in the DS1104 board is shown in Fig. 1. The stator currents and the DC-link voltage are measured, and the reference voltage obtained from the current controller is used for the observer. The sampling is synchronized to the modulation, and both the switching frequency and the sampling frequency are 5 kHz. A simple current feedforward compensation for dead times and power device voltage drops is applied. The control system shown in Fig. 1 is augmented with a speed controller, whose feedback signal is the speed estimate $\hat{\omega}_m$ obtained from the proposed observer. The bandwidth of this PI controller, including active damping [11], is $2\pi \cdot 5.3 \text{ rad/s}$ (0.05 p.u.). The stator-resistance adaptation is disabled in all experiments.

The saturation has been modeled as functions of the measured current,

$$L_d = \begin{cases} L_{d0} - \alpha i_d - \delta \frac{i_d}{i_{d0}} & \text{if } i_d \leq i_{d0} \\ L_{d0} - \alpha i_d - \delta \frac{i_d}{i_{d0}} & \text{otherwise} \end{cases}$$

$$L_q = \begin{cases} L_{q0} - \gamma \sqrt{|i_q|} - \delta \frac{i_q}{i_{q0}} & \text{if } i_q \leq i_{q0} \\ L_{q0} - \gamma \sqrt{|i_q|} - \delta \frac{i_q}{i_{q0}} & \text{otherwise} \end{cases}$$

where $i_{d0}$ and $i_{q0}$ are transition values for $i_d$ and $i_q$ to avoid divisions by small numbers. The measured inductances and the curves from the fitted functions are shown in Fig. 5. In Fig. 5(a), the $d$ axis inductance $L_d$ is shown as a function of $i_d$ for three different values of $i_q$. In Fig. 5(b), $L_q$ is shown as a function of $i_q$ for three different values of $i_d$. The saturation model parameters are: $L_{d0} = 3.15 \text{ p.u.}$, $L_{q0} = 0.685 \text{ p.u.}$, $\alpha = 2.24 \text{ p.u.}$, $\gamma = 0.353 \text{ p.u.}$ and $\delta = 0.085 \text{ p.u.}$.

V. EXPERIMENTAL RESULTS

Fig. 6 shows experimental results of a slow change of $i_d$ from 0.3 p.u. to 0.5 p.u. at the speed $\hat{\omega}_m = 0.1 \text{ p.u.}$ when the parameter $\kappa = \sqrt{3}$ and $-50\%$ of the rated load torque is applied. It can be seen that as $i_d$ increases in the regenerating mode, the noise in the position estimate increases. This behavior suggests that the noise originates from saturation-induced harmonics. The frequency of the noise is approximately 21 Hz, which is two times the operating-point frequency (10.5 Hz), as can be seen in the magnification in Fig. 6(b).

Results of a stepwise speed reversal from $\hat{\omega}_m = 0.1 \text{ p.u.}$ to $\hat{\omega}_m = -0.10 \text{ p.u.}$ and back to 0.10 p.u. are depicted in Fig. 7. The rated load torque is applied, and $i_d = 0.5 \text{ p.u.}$ It can be seen that the noise in the estimation error is amplified in the regenerating mode in Fig. 7(a), when $\kappa = \sqrt{3}$. With the selection (16) and $\kappa_{\text{min}} = 0.6$, this noise is reduced in Fig. 7(b), as assumed based on the eigenvector analysis.

The effect of the parameter errors on the position estimation error at the speed $\hat{\omega}_m = 0.1 \text{ p.u.}$ with 50% rated load torque
Fig. 6. Experimental results of a slow change in \(i_d\) from 0.3 p.u. to 0.5 p.u. with \(-50\%\) of the rated load torque applied when \(\kappa = \sqrt{3}\). (a) full sequence, (b) magnification from \(10 \text{ s} \leq t \leq 11 \text{ s}\).

Fig. 7. Experimental results of a stepwise speed reversal (0.10 p.u. \(\rightarrow\) \(-0.10\) p.u. \(\rightarrow\) 0.10 p.u.) with rated load torque applied: (a) \(\kappa = \sqrt{3}\), (b) \(\kappa\) is selected according to (16) and \(\kappa_{\text{min}} = 0.6\). The data is captured by disabling the saturation model and varying each model parameter from 90\% up to 110\% of the actual value in 10 seconds. It can be seen that the model parameters \(\hat{R}_s\) and \(\hat{L}_q\) have only a small effect on the position error, whereas an incorrect value for \(\hat{L}_d\) increases the estimation error rapidly. It should be noted that the relative errors of \(L_d\) and \(L_q\) are defined with respect to the (original) operating-point values. As the estimation error increases, the actual values of \(i_d\) and \(i_q\) change, resulting in changes in actual values of \(L_d\) and \(L_q\) due to saturation.

VI. CONCLUSIONS

In this paper, equations for steady-state errors and analytical stability conditions are derived for the reduced-order position observer with incorrect parameter estimates. Based on the stability conditions and small-signal dynamics, design rules
Fig. 8. Measured errors in the position estimate at $\hat{\omega}_m = 0.1$ p.u. with 50% rated load torque applied. The data is captured by varying each model parameter from 90% up to 110% of the actual value in 10 seconds.

...are proposed. The system can be augmented with the stator-resistance adaptation for improved low-speed operation. The equations for steady-state errors and small-signal dynamics are derived for the system augmented with the resistance adaptation. The performance of the proposed observer design was evaluated using laboratory experiments with a 6.7-kW SyRM drive. Stable operation at low speed under different parameter errors and different loads was demonstrated. With the proposed design rules, saturation-induced noise in the position estimate can be reduced.

ACKNOWLEDGMENT

The authors gratefully acknowledge ABB Oy for the financial support.

REFERENCES